

Control of the UltraLITE Precision Deployable Test Article Using Adaptive Spatio-Temporal Filtering Based Control

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- UltraLITE Deployable Optical Telescope program
- DOT test beds
 - Mirror Mass Simulator
 - PDOS
 - DOT BGD
- Active structural control issues
- Spatio-Temporal Filtering (STF)
- STF based structural control
- PDOS test experience



- Phase II SBIR award from Ballistic Missile Defense Organization (BMDO)
- Contract managed by and technical collaboration with Air Force Research Lab - Kirtland AFB



- Large aperture/resolution through deployable, sparse, optical array
- Deployable primary mirrors
- Golay 6 configuration
- Telescoping secondary tower

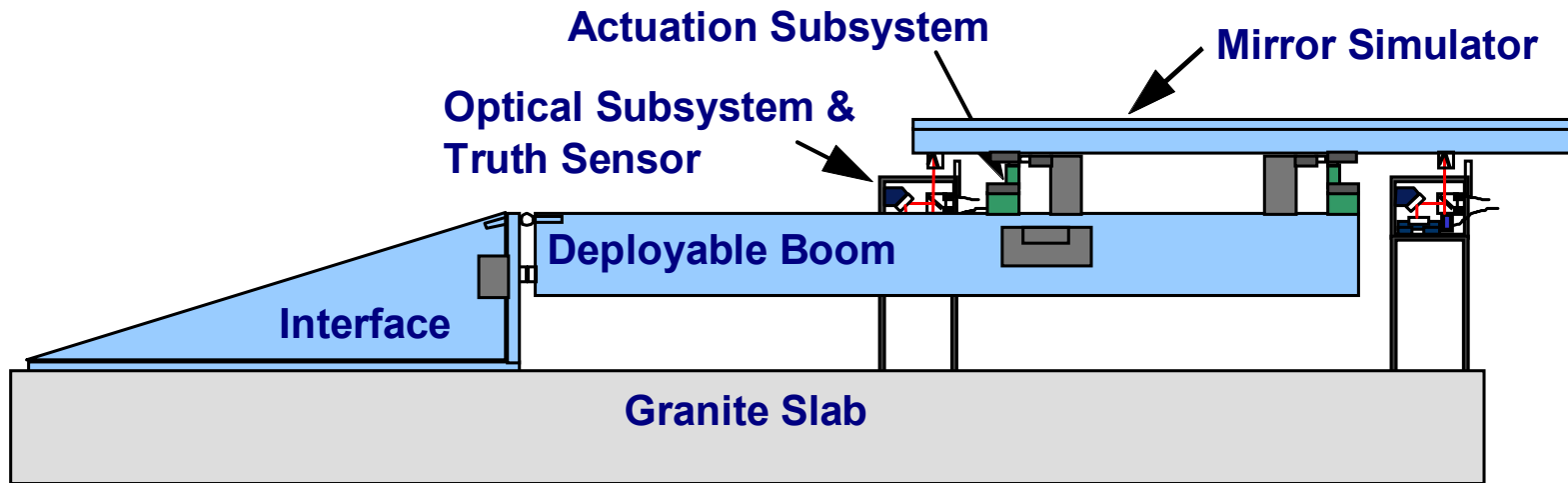




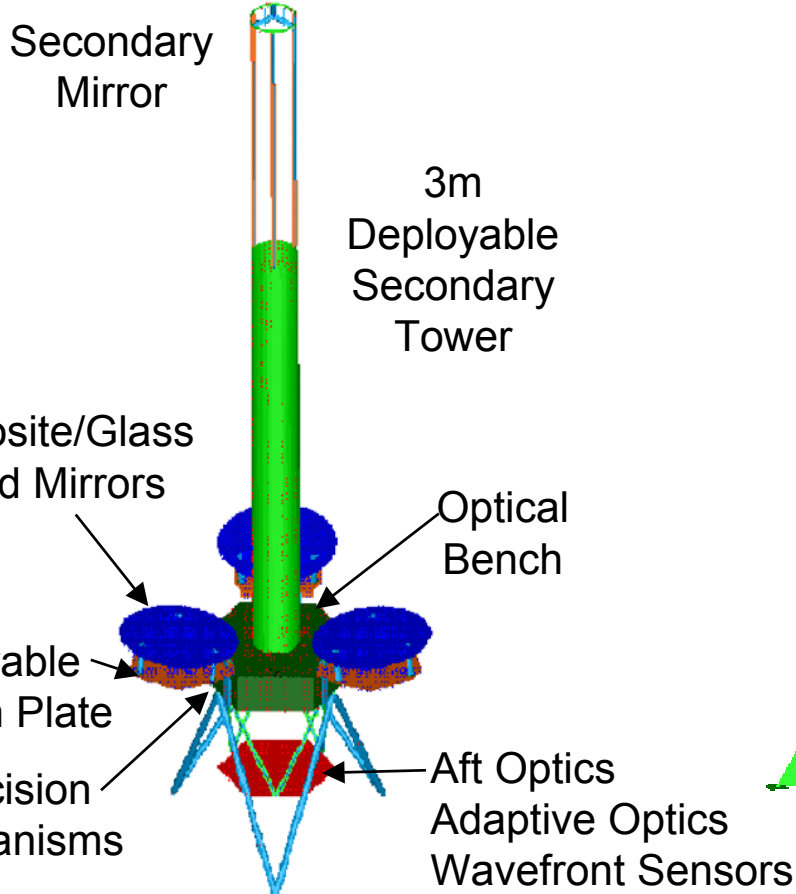
- Mirror Mass Simulator (MMS)
- Precision Deployable Optics Structure (PDOS)
- Deployable Optical Telescope Brassboard Ground Demonstration (BGD)

- Mirror Mass Simulator mounted to optics bench
- 3 interferometer displacement sensors
- 3 piezo stack actuators
- Electromagnetic disturbance shaker

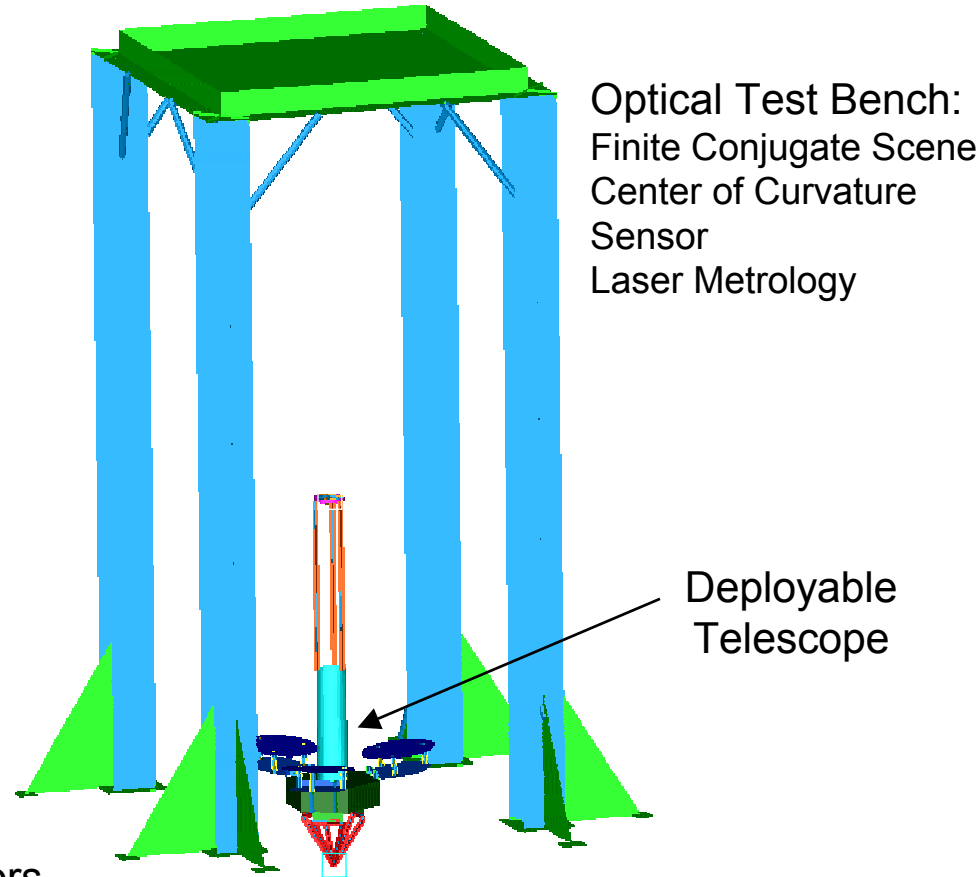




1.5m Deployable Telescope



Test Tower



Integrated Technology in Simulated Space/Ops Environment
Supporting SBL, Global Virtual Presence, and Tactical Imaging Missions



SDL's primary mission is to provide a Vibration Control System that will assist the Optical Control System in meeting the DOT mirror positioning requirements

- **Precision Deployable Optical Structure (PDOS):**

Achieve 30 nanometers or less RMS value for relative displacement between the granite slab and the mirror mass simulator

- **Deployable Optical Telescope (DOT)**

(1) Maintain the position of the primary mirror segments within:

Piston: ± 14 nanometers error per segment

Tilt: ± 95 nanoradians error per segment

(2) Maintain the position of the secondary mirror within:

Decenter ± 50 microns

Piston: ± 4 microns

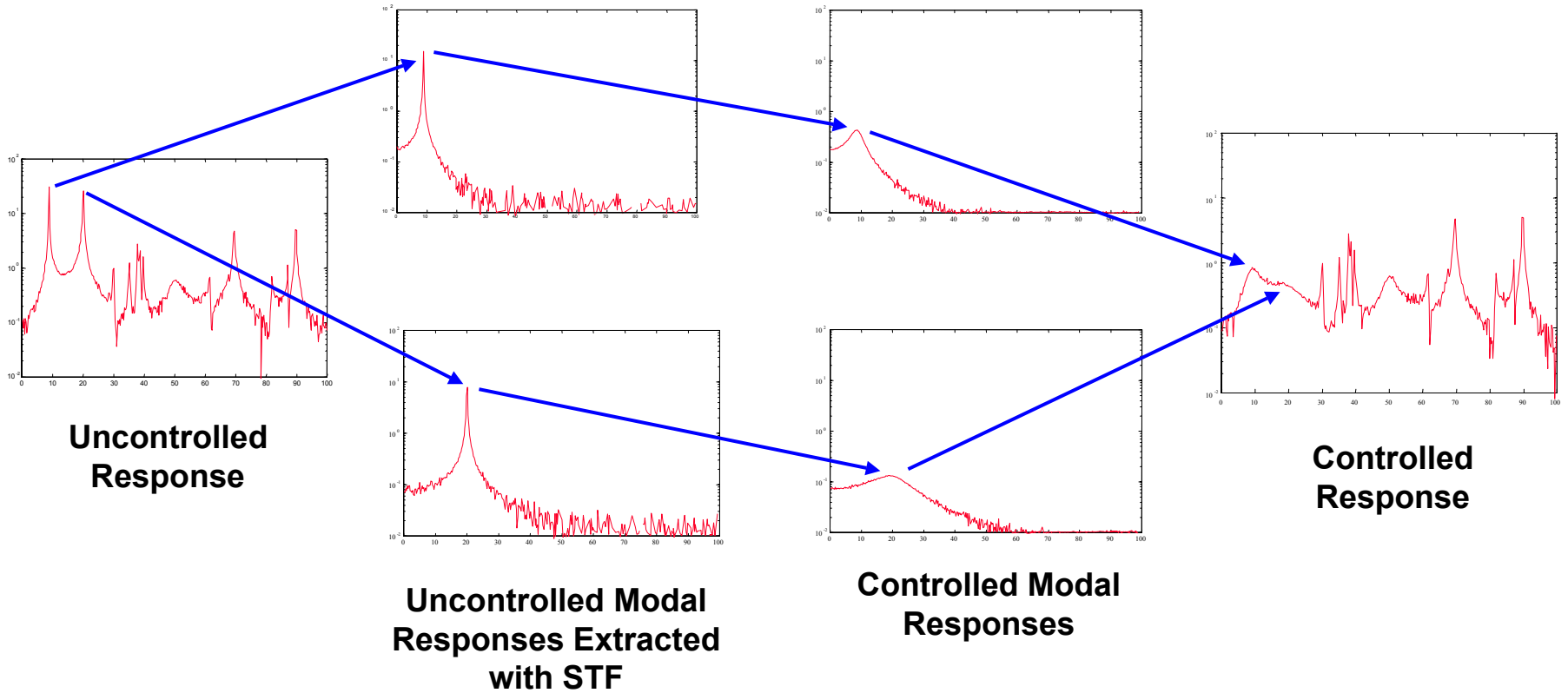
Tilt: ± 20 microradians



- Disturbances
 - torque wheel actuators
 - slewing
 - space based laser
- Vibration Control
 - isolation
 - passive vibration control
 - high bandwidth position control
 - active vibration control



- Modeling - accurate and complete dynamic models of complex “real-world” systems are difficult to obtain.
- Time Variance - Often, by the time you’ve got the model the system has changed - It’s a moving target.
 - System dynamics - temperature, load, wear, damage
 - Discrete failures - sensors, actuators, signal conditioning
- Computational burden





$$M\ddot{x} + C\dot{x} + Kx = f$$



Modal Coordinate Transformation

$$\begin{bmatrix} \cdot & & \\ & \cdot & \\ & & \cdot \end{bmatrix} m \ddot{\eta} + \begin{bmatrix} \cdot & & \\ & \cdot & \\ & & \cdot \end{bmatrix} c \dot{\eta} + \begin{bmatrix} \cdot & & \\ & \cdot & \\ & & \cdot \end{bmatrix} k \eta = \Phi^T f$$

$$x(t) = \sum_{r=1}^N \phi_r \eta_r(t) = \Phi \eta(t)$$



**Spatial filter
vector Ψ**

$$\begin{aligned}\Psi_i^T \phi_r &= 0 & i \neq r \\ &= 1 & i = r\end{aligned}$$

**Extract single mode
response from
measured response**

$$\begin{aligned}\Psi_i^T x(t) &= \Psi_i^T \sum_{r=1}^N [\phi_r \eta_r(t)] \\ &= \Psi_i^T \phi_i \eta_i(t) \\ &= \eta_i(t)\end{aligned}$$



**Spatial filter
estimate of η at
time k**

$$\hat{\eta}_k = \Psi^T x_k$$

**Spatio-Temporal filter
estimate of η at
time k**

$$\hat{\eta}_k = \Psi^T \left\{ \begin{array}{c} x_k \\ x_{k-1} \\ \vdots \\ x_{k-Nt} \end{array} \right\}$$



- FIR or “all-zero” filter on each channel
- Pole-zero cancellation & preferential pass filter
 - fewer sensors required
- Inherent estimation of modal velocity
- Compensation for filter delays, sensor & signal conditioning dynamics
- Non-homogeneous sensor suites - piezo patches, accelerometers, etc.



- Know only poles of controlled modes
- Don't know
 - mode shapes
 - modal scaling factors (modal mass)
 - modal participation vectors
 - anything about uncontrolled modes (not even poles)

SDL Adaptive Calculation of STF Coefficients using Reference Model Approach



SDOF (Single Mode) Reference Model

$$\eta_{k+1}^{(r)} = z_\lambda \eta_k^{(r)} + l^T f_k$$

$$\left. \begin{aligned} \eta_{k+1}^{(r1)} &= z_\lambda \eta_k^{(r1)} + f_k^{(1)} \\ &\vdots \\ \eta_{k+1}^{(rN_i)} &= z_\lambda \eta_k^{(rN_i)} + f_k^{(N_i)} \end{aligned} \right\} + \eta_k^{(r)} = l^T \begin{Bmatrix} \eta_k^{(r1)} \\ \vdots \\ \eta_k^{(rN_i)} \end{Bmatrix} = l^T \eta_k^r$$

SDL Adaptive Calculation of STF Coefficients using Reference Model Approach



$$\begin{aligned} e_k &= \eta_k^{(r)} - \hat{\eta}_k \\ &= l^T \eta_k^r - \psi^T \begin{Bmatrix} x_k \\ x_{k-1} \\ \vdots \\ x_{k-Nto} \end{Bmatrix} = \begin{Bmatrix} \psi \\ l \end{Bmatrix}^T \begin{Bmatrix} -x_k \\ \vdots \\ -x_{k-Nto} \\ \eta_k^r \end{Bmatrix} \end{aligned}$$



Control Command Vector for i'th mode

$$f_c^{(i)} = \hat{\eta}^{(i)} \alpha^{(i)} v^{(i)}$$

Modal Coordinate Velocity Estimate

Force Vector

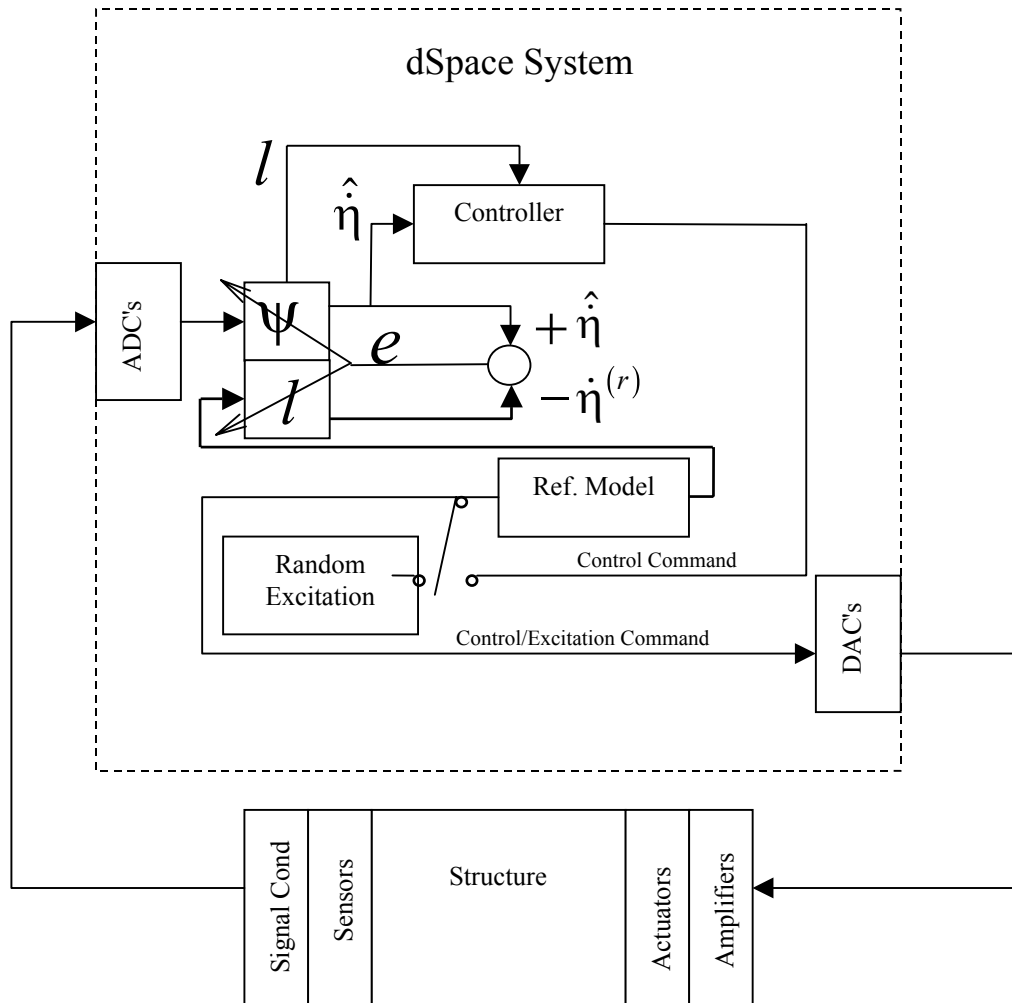
Scalar Feedback Gain

$$f_c^{(i)} = \hat{\eta}^{(i)} \alpha^{(i)} l^{(i)}$$

Estimated Modal Participation Vector is Ideal Force Vector

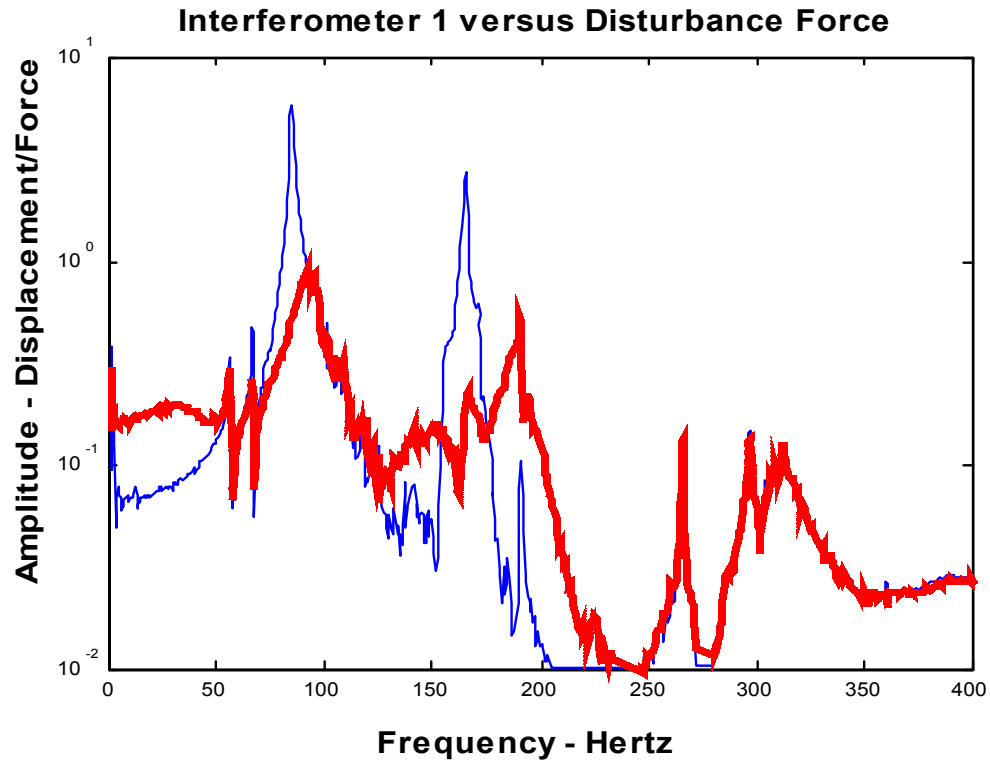


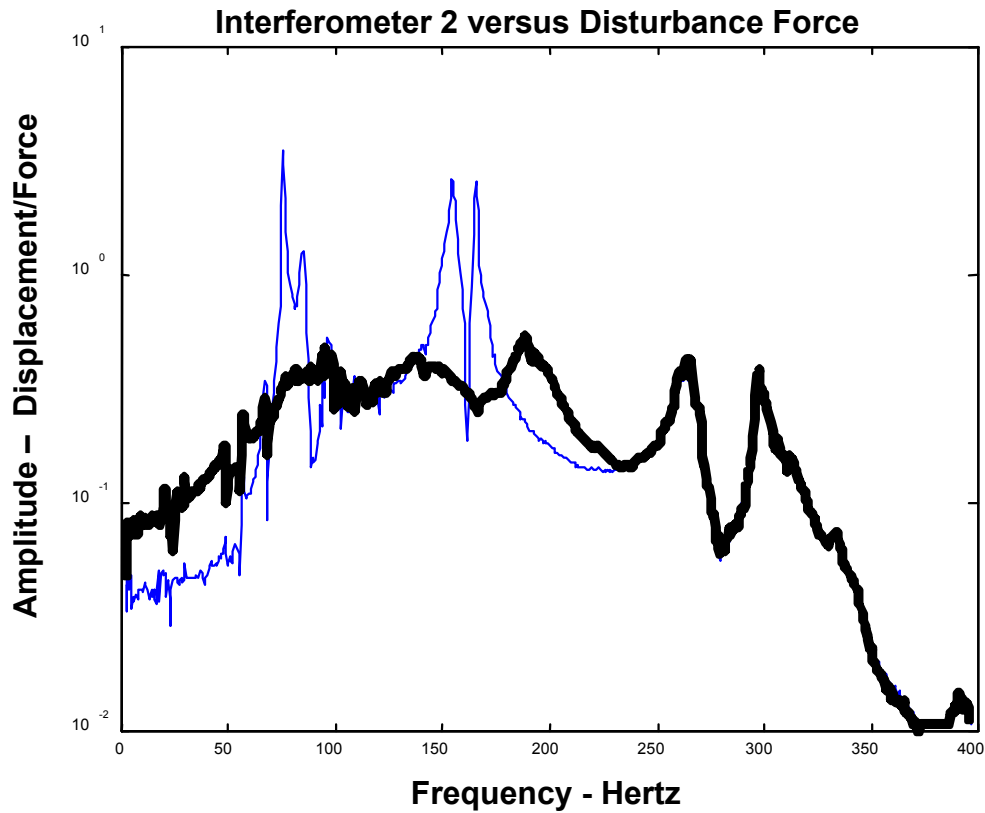
- STF based velocity feedback
- 3 inputs, 3 outputs, 5 controlled modes
- Random disturbance excitation
- 1 1/2 days to implement
 - familiarization with test bed
 - all system ID
 - control implementation and testing

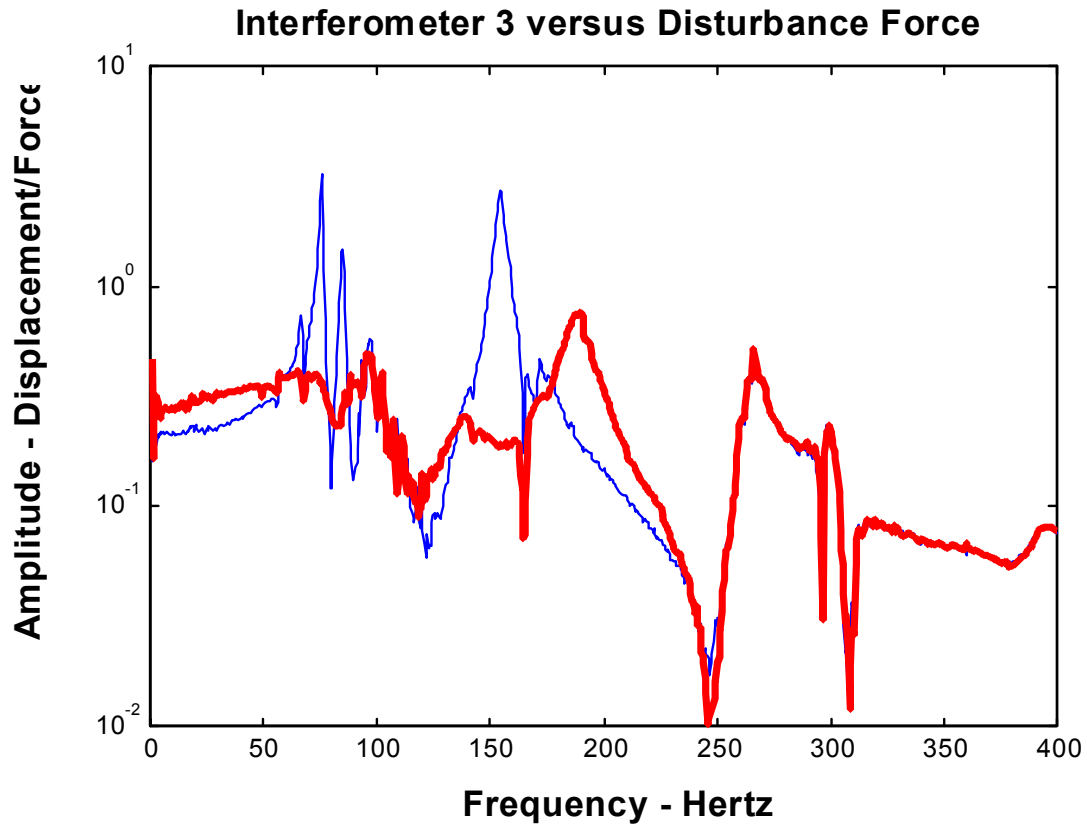


- Mirror Mass Simulator mounted to optics bench
- 3 interferometer displacement sensors
- 3 piezo stack actuators
- Electromagnetic disturbance shaker





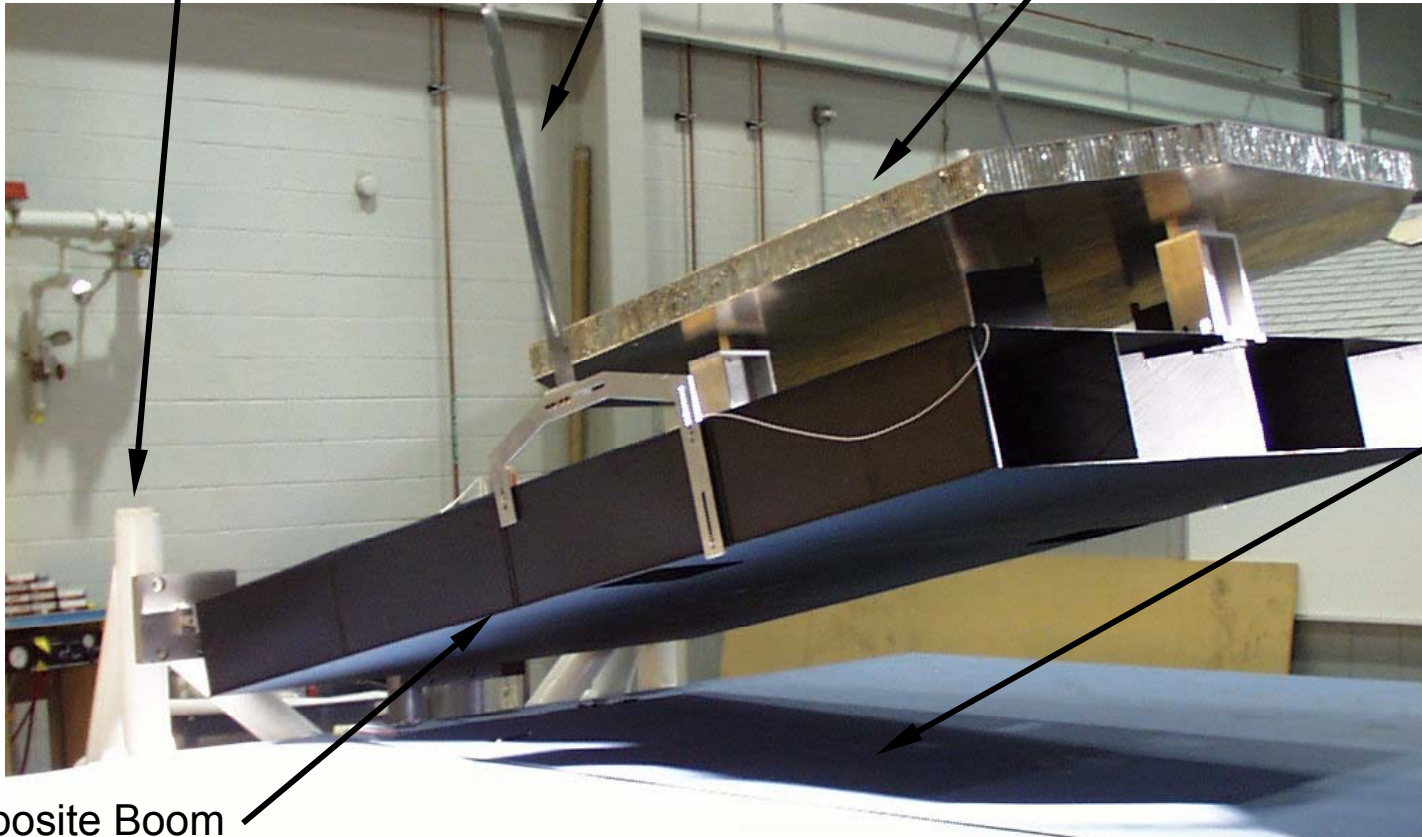




Back-Up Structure

Gravity Off-Load

Mirror Inertial Simulator



Reference Bench

Composite Boom

Proved Deployment, Acquisition, Maintenance and Control System for a 2m Optical Segment

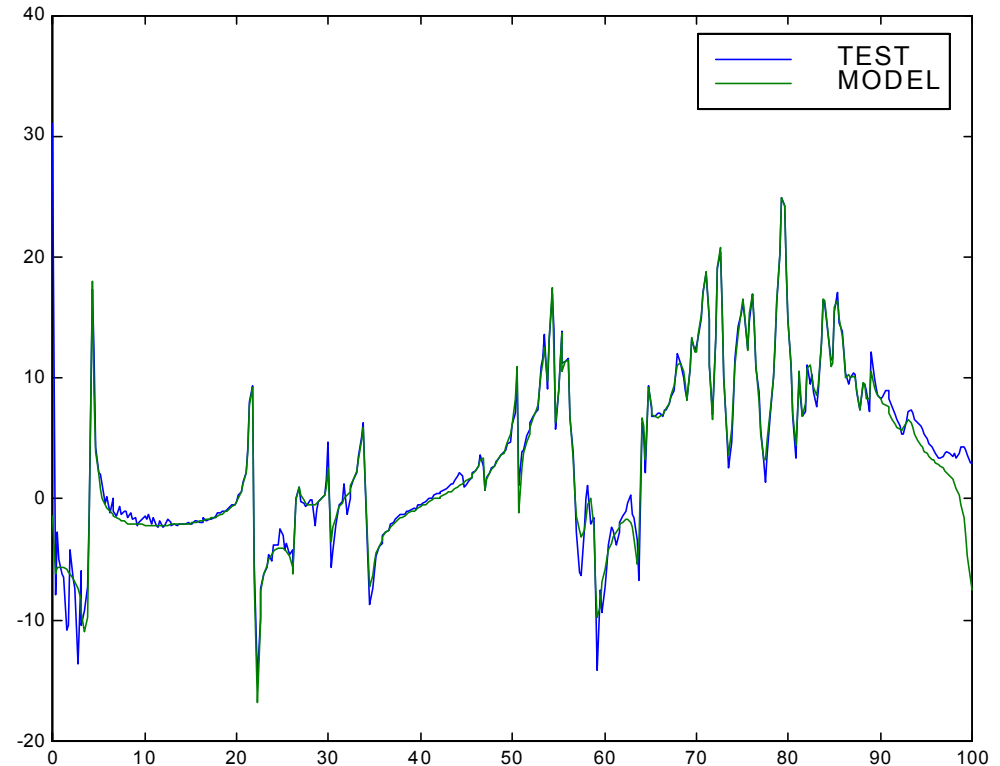




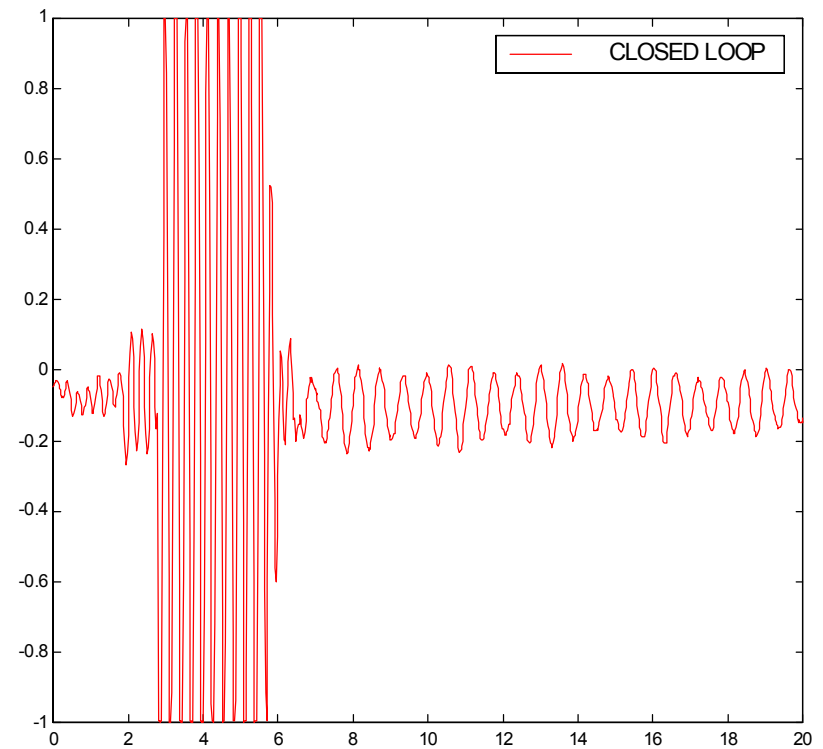
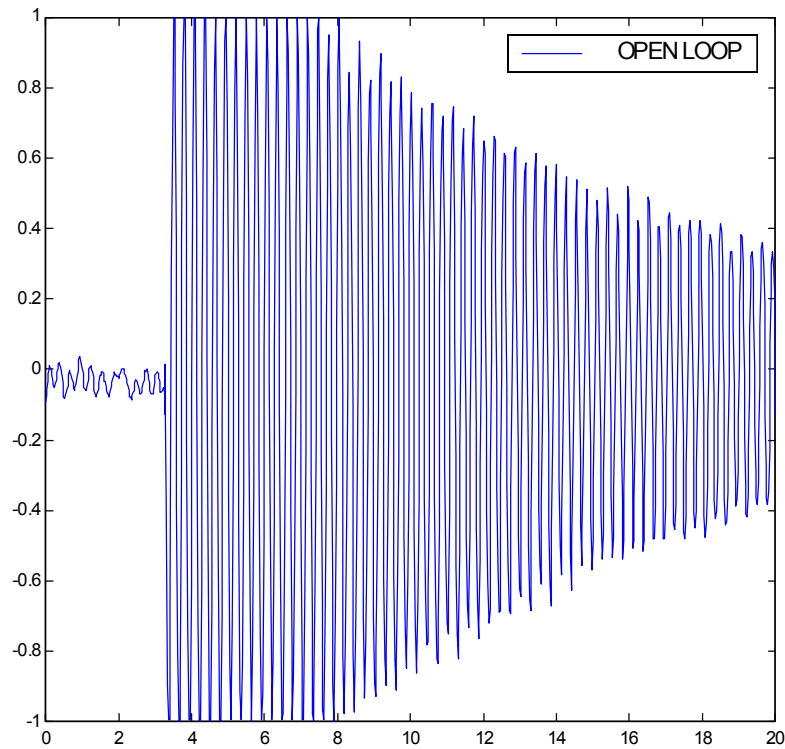


Interferometer 1 / PZT 2 FRF

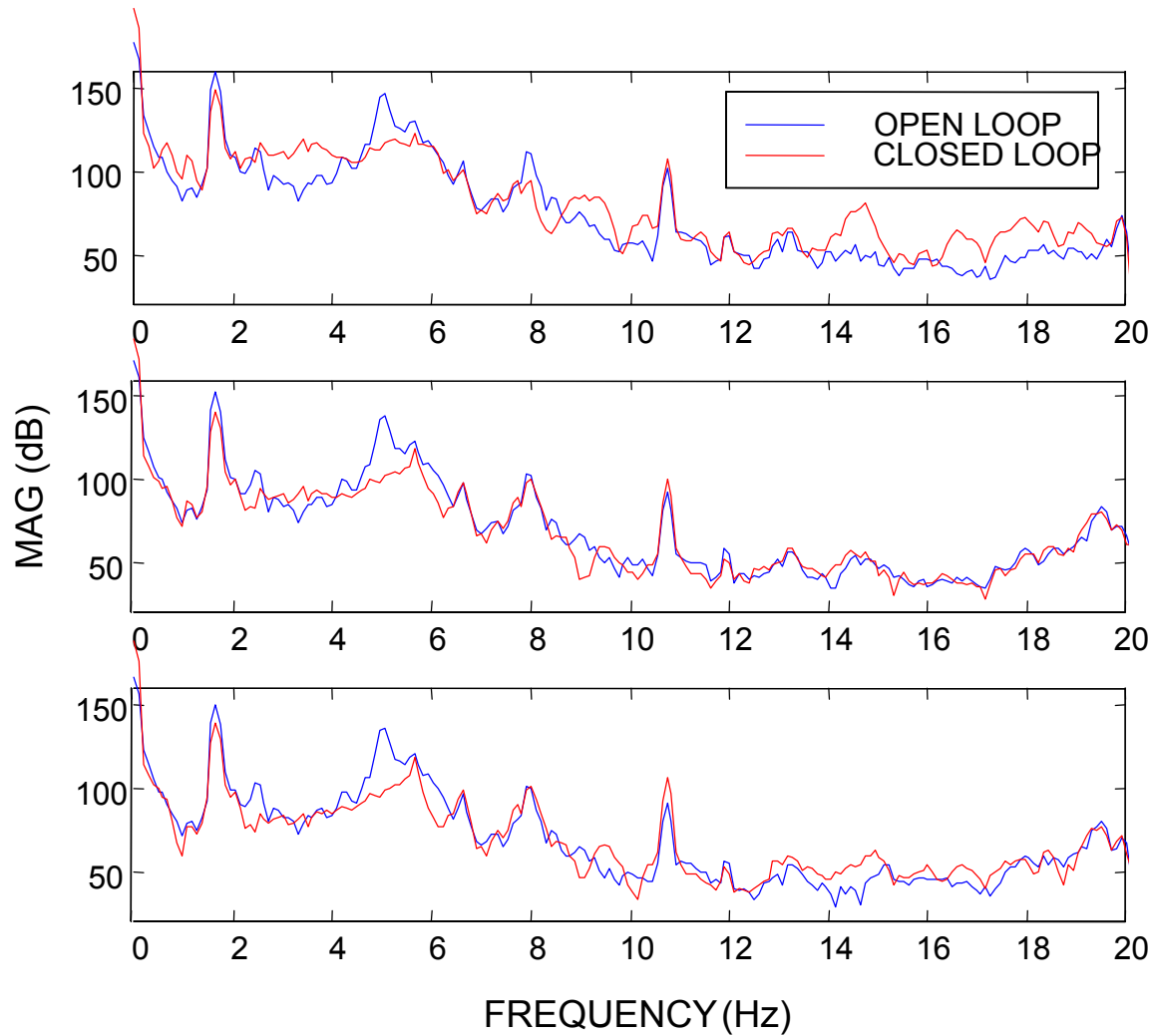
- 5 Hertz boom mode only mode in low frequency range
- 1.6 Hertz slab mode not apparent - must treat as a disturbance



STF Control PDOS 5 Hz Boom Mode



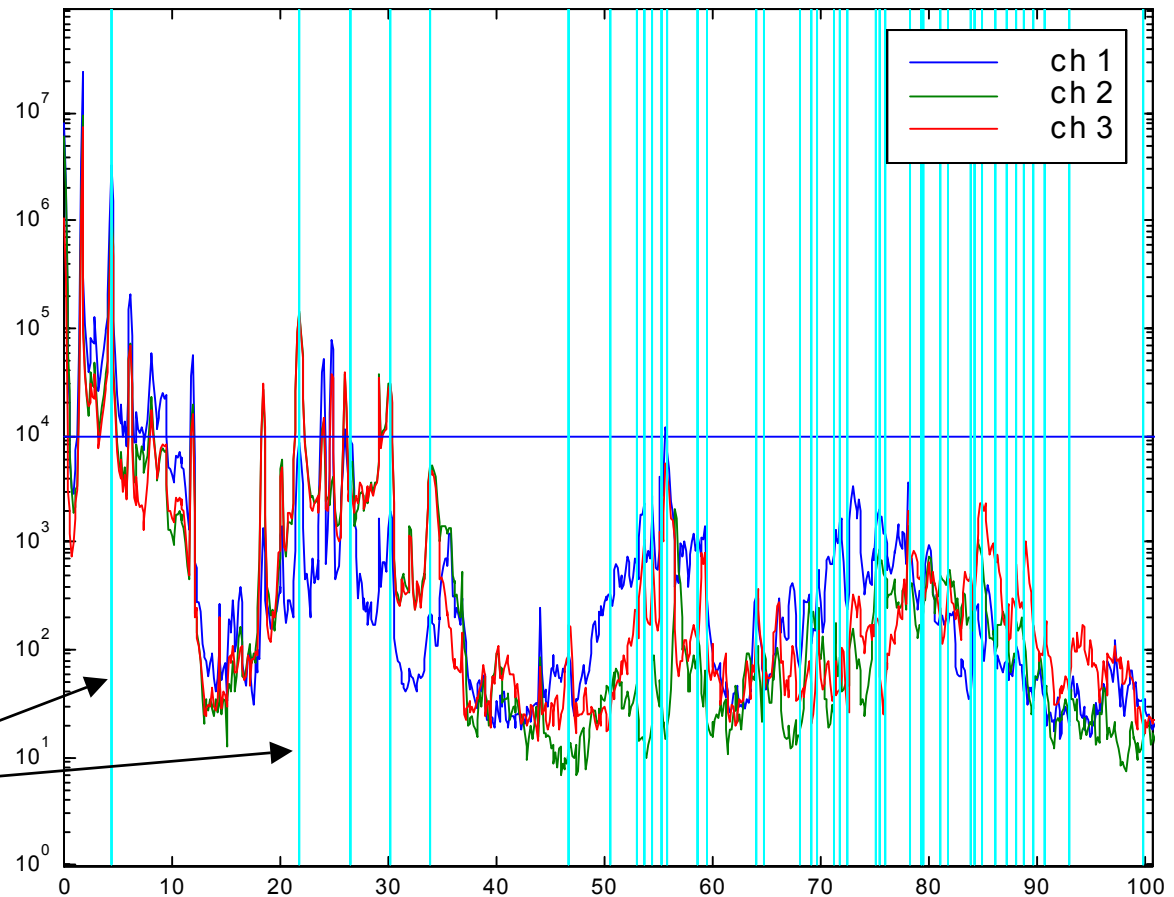
Open and Closed Loop Interferometer PSD's

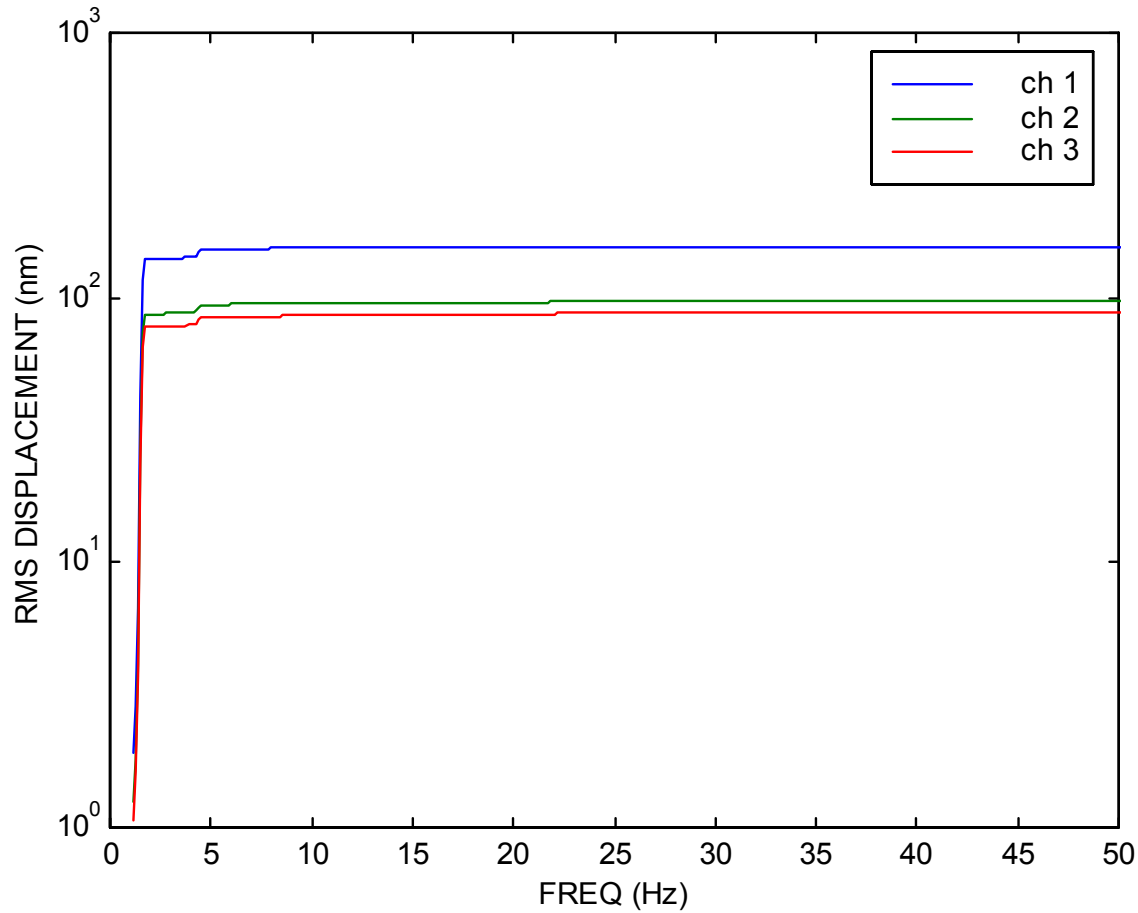




- RMS Vibration, 0-250 Hz
 - Int 1: 171-430 nm
 - Int 2: 117-264 nm
 - Int 2: 93-239 nm
- Resonant and forced vibration

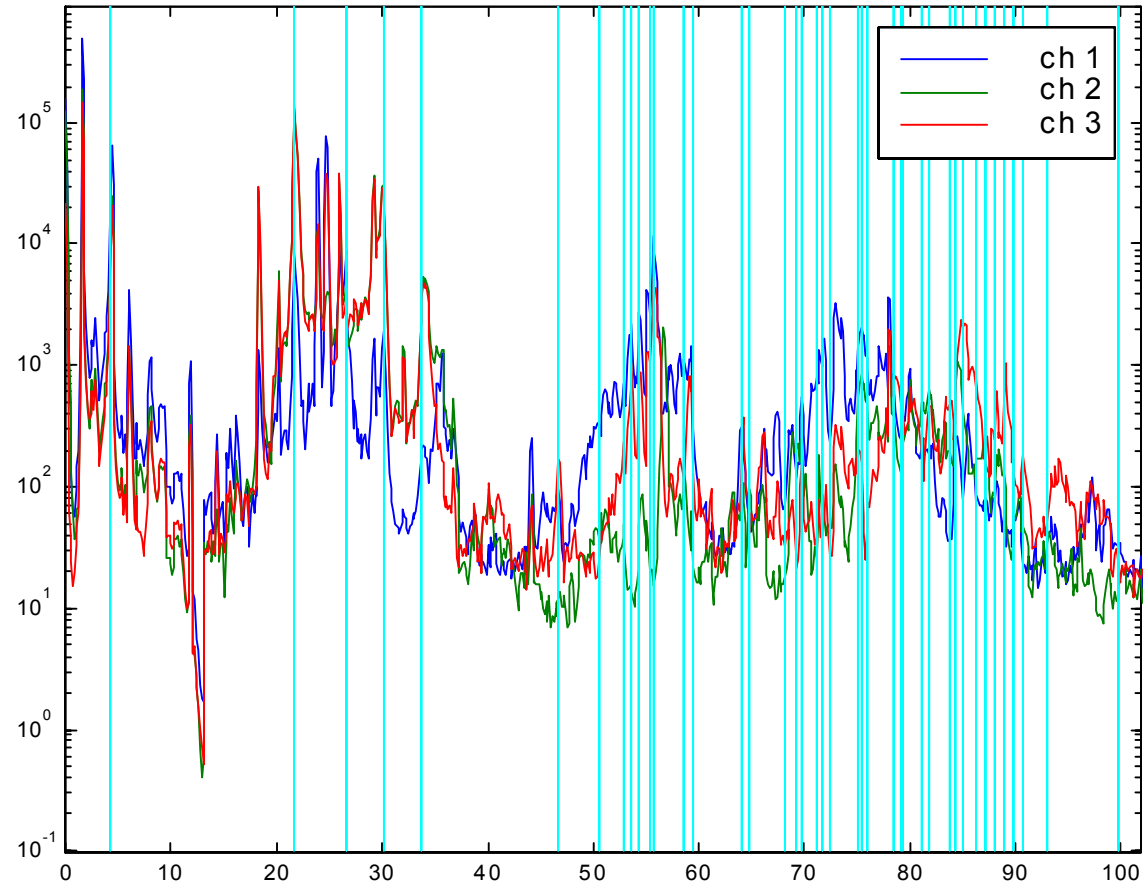
Resonant
Frequencies







- RMS Vibration, 0-250 Hz
- Assuming 50x reduction 0-13 Hz.
 - Int 1: 29.99 nm
 - Int 2: 28.51 nm
 - Int 3: 27.42 nm



- Note the control objective is 30 nm RMS vibration levels



- Resonant control alone is not sufficient to meet PDOS/DOT optical jitter control requirements
- “High” bandwidth position control in conjunction with resonant mode control required
- STF based modal control is practical approach for resonant mode control;
 - Implement effective MIMO control on complex, “real-world” structures with little knowledge of dynamics
 - Adapts to sensor/actuator failure
 - Accommodates filter/signal conditioning dynamics
 - Easily updated to accommodate changing system dynamics (only update poles of controlled modes)